

Adaptive PID Speed Control Design for Permanent Magnet Synchronous Motor Drives

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Abstract—This paper proposes an adaptive proportional-integral-derivative (PID) speed control scheme for permanent magnet synchronous motor (PMSM) drives. **The proposed controller consists of three control terms: a decoupling term, a PID term, and a supervisory term. The first control term is employed to compensate for the nonlinear factors, the second term is made to automatically adjust the control gains, and the third one is designed to guarantee the system stability.** Different from the offline-tuning PID controllers, **the proposed adaptive controller includes adaptive tuning laws to online adjust the control gains based on the gradient descent method.** Thus, it can adaptively deal with any system parameter uncertainties in reality. The proposed scheme is not only simple and easy to implement, but also it guarantees an accurate and fast speed tracking. It is proven that the control system is asymptotically stable. To confirm the effectiveness of the proposed algorithm, the comparative experiments between the proposed adaptive PID controller and the conventional PID controller are performed on the PMSM drive. Finally, it is validated that the proposed design scheme accomplishes the superior control performance (faster transient response and smaller steady-state error) compared to the conventional PID method in the presence of parameter uncertainties.

Index Terms—Adaptive control, parameter uncertainties, proportional-integral-derivative (PID) control, surface-mounted permanent magnet synchronous motor (SPMSM).

I. INTRODUCTION

IN recent years, the ac motors are extensively applied in home appliances as well as industrial applications such as electric vehicles, wind generation systems, industrial robots, air conditioners, washing machines, etc. There are two main categories of the ac motors: induction motors (IMs) and permanent magnet synchronous motors (PMSMs). Nowadays, the IMs are used in about 70% of industrial electric motors due to their simplicity, ruggedness, and low production costs [1]–[5]. Despite that, the PMSMs are gradually taking over the IMs owing to their high efficiency, low maintenance cost, and high power density.

Manuscript received January 6, 2014; revised March 5, 2014; accepted March 6, 2014. Date of publication March 12, 2014; date of current version October 7, 2014. This work was supported by the National Research Foundation of Korea (NRF) under Grant 2012R1A2A2A01045312 funded by the Korea government (MSIP, Ministry of Science, ICT & Future Planning). This work was also supported by Basic Science Research Program through the NRF funded by the Ministry of Education (2012R1A1A2001439). Recommended for publication by Associate Editor J. O. Ojo.

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Digital Object Identifier 10.1109/TPEL.2014.2311462

However, the PMSM system is not easy to control because it is a nonlinear multivariable system and its performance can be highly affected by parameters variations in the run time [6]–[9]. Therefore, researchers always desire to design a high-performance controller which has a simple algorithm, fast response, high accuracy, and robustness against the motor parameter and load torque variations.

Traditionally, the proportional-integral-derivative (PID) controller is widely adopted to control the PMSM systems in industrial applications owing to its simplicity, clear functionality, and effectiveness [10]. However, a big problem of the traditional PID controller is its sensitivity to the system uncertainties. Thus, the control performance of the conventional PID method can be seriously degraded under parameter variations. Some groups of researchers try to overcome this disadvantage by proposing the hybrid PID controllers or new tuning rules [11]–[13]. In [11], a hybrid control system, which contains a fuzzy controller in the transient and a PI controller in the steady state, is proposed. In [12], the fuzzy rules are employed for tuning the PI gains. Unfortunately, both these methods use offline-tuning rules, which lack the adaptability to deal with the time-varying system uncertainties. An adaptive PI controller with an online-tuning rule is presented in [13]. Although this controller does not require the exact knowledge of any motor parameter, the authors do not show the results under parameter uncertainties.

Recently, many researchers have presented various advanced control strategies to efficiently control the PMSM systems, such as fuzzy logic control (FLC), nonlinear optimal control (NOC), sliding mode control (SMC), neural network control (NNC), adaptive control, etc. The FLC [14], [15] is a preferred research topic due to its fuzzy reasoning capacity. However, as the number of the fuzzy rules increases, the control accuracy can get better but the control algorithm can be complex. The NOC is successfully applied on the PMSM drives [16], [17]. Unfortunately, this control method requires full knowledge of the motor parameters with a sufficient accuracy and the results under serious variations of the mechanical parameters are not shown. The SMC has achieved much popularity in the speed control of the PMSM drives because of its great properties such as robustness to external load disturbances and fast dynamic response [6], [18]–[21]. However, its system dynamics are still subject to the parameter variations and chattering problem. Meanwhile, the NNC technique has been presented as a substitutive design method to control the speed of the PMSM system [22]–[24]. The most valuable property of this technique is its ability to approximate the linear or nonlinear mapping through learning. However, the high computational burden increases

the complexity in the control algorithm, which limits the implementation of this strategy in the practical applications. Next, the adaptive control is also an interesting method for the PMSM drives because it can deal with the motor parameter and load torque variations [25], [26]. Nevertheless, in these two papers, only the stator inductances and load torque variations are considered. The authors neglect the uncertainties of other motor parameters such as stator resistance, moment of inertia, viscous friction coefficient, etc. Moreover, the adaptive control algorithm [26] does not guarantee the convergence condition of the system dynamic error.

By combining the simplicity and effectiveness of the traditional PID control and the automatic adjustment capability of the adaptive control, this paper proposes a simple adaptive PID control algorithm for the PMSM drives. The adaptive PID controller encompasses the adaptive tuning laws which are designed to online adjust the control gains by using the supervisory gradient descent method. Therefore, when the motor parameters vary, the PID gains are automatically tuned to attain their optimal values. Consequently, the proposed control system achieves a good regulation performance such as fast dynamic response and small steady-state error even under system parameter uncertainties. The stability analysis of the proposed control strategy is described in details through the Lyapunov stability theories. The experimental results demonstrate the validity and feasibility of the proposed adaptive PID control method in comparison with the conventional PID control scheme under parameter uncertainties.

II. SYSTEM MODEL DESCRIPTION AND THE DYNAMIC ERROR SYSTEM

A. System Model Description

The mathematical model of a surface-mounted permanent magnet synchronous motor (SPMSM) drives can be described by the following equations in a dq synchronously rotating reference frame:

$$\begin{aligned}\dot{\omega} &= \frac{3}{2J} \frac{p^2}{4} \psi_m i_{qs} - \frac{B}{J} \omega - \frac{p}{2J} T_L \\ \dot{i}_{qs} &= -\frac{R_s}{L_s} i_{qs} - \frac{\psi_m}{L_s} \omega + \frac{1}{L_s} V_{qs} - \omega i_{ds} \\ \dot{i}_{ds} &= -\frac{R_s}{L_s} i_{ds} + \frac{1}{L_s} V_{ds} + \omega i_{qs}\end{aligned}\quad (1)$$

where ω is the electrical rotor speed, i_{ds} and i_{qs} are the d -axis and q -axis stator currents, V_{ds} and V_{qs} are the d -axis and q -axis voltage inputs, L_s is the stator inductance, R_s is the stator resistance, ψ_m is the magnetic flux linkage, J is the moment of inertia, B is the viscous friction coefficient, p is the number of poles, and T_L is the load torque.

Depending on R_s , L_s , J , B , and ψ_m , the system parameters $k_1 - k_6$ can be denoted as

$$\begin{aligned}k_1 &= \frac{3}{2J} \frac{p^2}{4} \psi_m, k_2 = \frac{B}{J}, k_3 = \frac{p}{2J} \\ k_4 &= \frac{R_s}{L_s}, k_5 = \frac{\psi_m}{L_s}, k_6 = \frac{1}{L_s}.\end{aligned}\quad (2)$$

Then, the SPMSM drive system model is rewritten as the following equations:

$$\begin{aligned}\dot{\omega} &= k_1 i_{qs} - k_2 \omega - k_3 T_L \\ \dot{i}_{qs} &= -k_4 i_{qs} - k_5 \omega + k_6 V_{qs} - \omega i_{ds} \\ \dot{i}_{ds} &= -k_4 i_{ds} + k_6 V_{ds} + \omega i_{qs}.\end{aligned}\quad (3)$$

B. Conventional PID Controller With Decoupling Term

First, the speed error (ω_e) and rotor acceleration (β) are defined as

$$\begin{aligned}\omega_e &= \omega - \omega_d \\ \beta &= \dot{\omega} = k_1 i_{qs} - k_2 \omega - k_3 T_L\end{aligned}\quad (4)$$

where ω_d is the desired speed.

From (3) and (4), the following dynamic equations can be derived:

$$\begin{aligned}\dot{\omega}_e &= \dot{\omega} - \dot{\omega}_d \\ \dot{\beta} &= k_1 (-k_4 i_{qs} - k_5 \omega + k_6 V_{qs} - \omega i_{ds}) - k_2 \dot{\omega} - k_3 \dot{T}_L \\ \dot{i}_{ds} &= -k_4 i_{ds} + k_6 V_{ds} + \omega i_{qs}.\end{aligned}\quad (5)$$

In practical applications, the desired speed and the load torque vary slowly in the sampling period. Thus, it can be reasonably supposed that the derivatives of ω_d and T_L can be neglected. Then, the system model (1) can be rewritten as

$$\begin{aligned}\dot{\omega}_e &= \beta \\ \dot{\beta} &= -k_2 \beta - k_1 k_4 i_{qs} - k_1 k_5 \omega - k_1 \omega i_{ds} + k_1 k_6 V_{qs} \\ \dot{i}_{ds} &= -k_4 i_{ds} + \omega i_{qs} + k_6 V_{ds}.\end{aligned}\quad (6)$$

Then, the second-order system can be achieved in the following:

$$\begin{aligned}\ddot{\omega}_e + \lambda \dot{\omega}_e &= -k_2 \beta - k_1 k_4 i_{qs} - k_1 k_5 \omega - k_1 \omega i_{ds} + \lambda \beta \\ \dot{i}_{ds} &= -k_4 i_{ds} + \omega i_{qs} + k_6 V_{ds}\end{aligned}\quad (7)$$

where λ is the positive control parameter.

Based on the basic theory of the feedback linearization control, the decoupling control term $u_f = [u_{1f} \ u_{2f}]^T$ is chosen as

$$\begin{aligned}u_{1f} &= (k_1 k_4 i_{qs} + k_1 k_5 \omega + k_1 \omega i_{ds} + (k_2 - \lambda) \beta) / (k_1 k_6) \\ u_{2f} &= (k_4 i_{ds} - \omega i_{qs}) / k_6.\end{aligned}\quad (8)$$

From (7) and (8), the dynamic error system can be formulated as follows:

$$\begin{aligned}\ddot{\omega}_e &= -\lambda \dot{\omega}_e + k_1 k_6 (V_{qs} - u_{1f}) \\ \dot{i}_{ds} &= k_6 (V_{ds} - u_{2f}).\end{aligned}\quad (9)$$

Then, the conventional PID controller is given by

$$V_{dq} = \begin{bmatrix} k_1 k_6 V_{qs} \\ k_6 V_{ds} \end{bmatrix} = B \begin{bmatrix} V_{qs} \\ V_{ds} \end{bmatrix} = u_f + u_{PID} \quad (10)$$

where $B = \text{diag}[k_1 k_6, k_6]$, u_f is the decoupling control term to compensate for the nonlinear factors, and u_{PID} is the PID

control term

$$u_{PID} = \begin{bmatrix} u_{1PID} \\ u_{2PID} \end{bmatrix} = \begin{bmatrix} -K_{1P}\omega_e - K_{1I} \int_0^t \omega_e dt - K_{1D} \frac{d\omega_e}{dt} \\ -K_{2P}i_{dse} - K_{2I} \int_0^t i_{dse} dt \end{bmatrix} \\ = EK \quad (11)$$

where (K_{1P}) , (K_{2P}) , (K_{1I}) , (K_{2I}) , and (K_{1D}) are the proportional gains, integral gains, and derivative gain of the PID control term, respectively. The state and gain matrices are given as

$$E = \begin{bmatrix} \int_0^t \omega_e dt & \omega_e & \beta & 0 & 0 \\ 0 & 0 & 0 & \int_0^t i_{ds} dt & i_{ds} \end{bmatrix} \\ K = [-K_{1I} \quad -K_{1P} \quad -K_{1D} \quad -K_{2I} \quad -K_{2P}]^T.$$

It should be noted that the derivative of the stator current is normally very noisy; thus, it is not included in (11).

III. PROPOSED ADAPTIVE PID CONTROLLER DESIGN

The conventional PID controller (10) with the offline-tuned control gains can give a good control performance if the motor parameters (k_1 to k_6) are accurately known. However, the system parameters gradually change during operating time; therefore, after a long running time, the control performance can be seriously degraded if changed system parameters are not updated. To overcome this challenge, this section presents the adaptive tuning laws for autoadjustment of the control gains. On that note, the control gains, denoted as K_{1I} , K_{1P} , K_{1D} , K_{2I} , and K_{2P} in (11), are adjusted to the proper values based on the supervisory gradient descent method. The proposed adaptive PID controller is assumed to have the following form:

$$V_{dqs} = u_f + u_{PID} - u_{PID0} + u_S \quad (12)$$

where u_f is the decoupling control term which compensates for the nonlinear factors as shown in (6), u_{PID} is the PID control term which includes the adaptive tuning laws, u_S is the supervisory control term which guarantees the system stability, and $u_{PID0} = EK_0$ (with $K_0 = [-K_{1I0} \quad -K_{1P0} \quad -K_{1D0} \quad -K_{2I0} \quad -K_{2P0}]^T$ is a constant coefficient matrix).

A. Proposed Adaptive PID Controller

In order to derive the proper adaptation laws, a new tracking error vector based on the reduced-order sliding mode dynamics is defined as

$$s(t) = \begin{bmatrix} s_1(t) \\ s_2(t) \end{bmatrix} = \begin{bmatrix} \lambda\omega_e + \beta \\ i_{ds} \end{bmatrix}. \quad (13)$$

Then, the transfer function $G(p)$ from s_1 to ω_e is given by the following strictly positive real function:

$$G(p) = \frac{\omega_e}{s_1} = \frac{1}{(\lambda + p)} \quad (14)$$

where p is the Laplace variable. Hence, it can be concluded that ω_e converges to zero as $s \rightarrow 0$.

From the viewpoints of the SMC method, the sliding condition that ensures the hitting and existence of a sliding mode

is deduced according to the Lyapunov stability theory. Commonly, the Lyapunov function candidate for the SMC is given by $V_1 = s^T s/2$. Then, the sliding condition can be obtained from the Lyapunov stability theory as

$$\dot{V}_1(t) = s^T \dot{s} < 0. \quad (15)$$

The sliding condition (15) guarantees that $s \rightarrow 0$ as $t \rightarrow \infty$. In order to derive the adaptive tuning laws for the PID gains, the supervisory gradient descent method is used to minimize the sliding condition $s^T \dot{s}$ in (15). The gradient descent search algorithm is calculated in the direction opposite to the energy flow, and the convergence properties of the PID gains can also be obtained. Therefore, the adaptation laws for the five control gains K_{1P} , K_{1I} , K_{1D} , K_{2P} , and K_{2I} can be easily obtained based on the supervisory gradient method as follows:

$$\begin{aligned} \dot{K}_{1P} &= -\gamma_{1P} \frac{\partial V_1}{\partial K_{1P}} = -\gamma_{1P} \frac{\partial V_1}{\partial u_{1PID}} \frac{\partial u_{1PID}}{\partial K_{1P}} = -\gamma_{1P} s_1 \omega_e \\ \dot{K}_{1I} &= -\gamma_{1I} \frac{\partial V_1}{\partial K_{1I}} = -\gamma_{1P} \frac{\partial V_1}{\partial u_{1PID}} \frac{\partial u_{1PID}}{\partial K_{1I}} \\ &= -\gamma_{1I} s_1 \int_0^t \omega_e dt \\ \dot{K}_{1D} &= -\gamma_{1D} \frac{\partial V_1}{\partial K_{1D}} = -\gamma_{1D} \frac{\partial V_1}{\partial u_{1PID}} \frac{\partial u_{1PID}}{\partial K_{1D}} = -\gamma_{1D} s_1 \beta \\ \dot{K}_{2P} &= -\gamma_{2P} \frac{\partial V_1}{\partial K_{2P}} = -\gamma_{2P} \frac{\partial V_1}{\partial u_{2PID}} \frac{\partial u_{2PID}}{\partial K_{2P}} = -\gamma_{2P} s_2 i_{ds} \\ \dot{K}_{2I} &= -\gamma_{2I} \frac{\partial V_1}{\partial K_{2I}} = -\gamma_{2I} \frac{\partial V_1}{\partial u_{2PID}} \frac{\partial u_{2PID}}{\partial K_{2I}} \\ &= -\gamma_{2I} s_2 \int_0^t i_{ds} dt \end{aligned} \quad (16)$$

where γ_{1P} , γ_{1I} , γ_{1D} , γ_{2P} , and γ_{2I} are the positive learning rates.

The adaptive tuning laws (16) can be expressed in the following vector form:

$$\dot{K} = \Phi E^T s \quad (17)$$

where $\Phi = \text{diag}(\gamma_{1I}, \gamma_{1P}, \gamma_{1D}, \gamma_{2I}, \gamma_{2P})$.

Remark 1: By utilizing the online-tuning rules (16), the control gains are automatically adjusted as the system parameters vary. Therefore, the proposed adaptive PID controller can overcome the disadvantage of all offline-tuning methods [11], [12] and can exhibit the good performance regardless of the system parameter uncertainties.

Next, the supervisory control term in (12) is necessary for pulling back the dynamic errors to the predetermined bounded region and guaranteeing the system stability. Assume that there exists an optimal PID control term (u_{PID}^*) such that

$$u_{PID}^* = u_{PID0} + \varepsilon \quad (18)$$

where $K^* = [-K_{1I}^* \quad -K_{1P}^* \quad -K_{1D}^* \quad -K_{2I}^* \quad -K_{2P}^*]^T$ is the optimal gain matrix, $\varepsilon = [\varepsilon_1 \varepsilon_2]^T$; ε_1 and ε_2 denote the approximation errors and they are assumed to be bounded by $0 \leq |\varepsilon_1| \leq \delta_1$ and $0 \leq |\varepsilon_2| \leq \delta_2$ in which δ_1 and δ_2 are the positive constants.

Then, the supervisory control term is designed as

$$u_S = \begin{bmatrix} -\delta_1 \cdot \text{sgn}(s_1) \\ -\delta_2 \cdot \text{sgn}(s_2) \end{bmatrix}. \quad (19)$$

To this end, the desired controller is obtained by combining the decoupling control term (8), PID control term (11) with the adaptation laws (16), and supervisory control term (19), as $V_{dqs} = u_f + u_{PID} + u_s$.

B. Stability Analysis

To analyze the stability of the dynamic error system, the following theorem is established.

Theorem: Consider the dynamic error system represented by (9). If the adaptive PID speed controller (12) with the adaptive tuning laws (16) is applied to (9), then the dynamic error system is asymptotically stable.

Proof: The following equation can be derived from (9), (12), and (18):

$$\begin{aligned} \dot{s} &= B(u_{PID} + u_S - u_{PID}^* + u_{PID}^* - u_{PID0}) \\ &= B(E\tilde{K} + u_S - \varepsilon) \end{aligned} \quad (20)$$

where $\tilde{K} = K - K^*$.

Let us define the errors of the control gains as follows:

$$\begin{aligned} \tilde{K}_{1P} &= K_{1P} - K_{1P}^*, \tilde{K}_{1I} = K_{1I} - K_{1I}^*, \tilde{K}_{1D} = K_{1D} - K_{1D}^* \\ \tilde{K}_{2P} &= K_{2P} - K_{2P}^*, \tilde{K}_{2I} = K_{2I} - K_{2I}^*. \end{aligned} \quad (21)$$

Based on (20) and (21), the following Lyapunov function candidate is chosen:

$$\begin{aligned} V_2(t) &= \frac{1}{2} s^T B^{-1} s + \frac{1}{\gamma_{1P}} \tilde{K}_{1P}^2 + \frac{1}{\gamma_{1I}} \tilde{K}_{1I}^2 + \frac{1}{\gamma_{1D}} \tilde{K}_{1D}^2 \\ &\quad + \frac{1}{\gamma_{2P}} \tilde{K}_{2P}^2 + \frac{1}{\gamma_{2I}} \tilde{K}_{2I}^2. \end{aligned} \quad (22)$$

The time derivative of the Lyapunov function $V_2(t)$ is given by

$$\begin{aligned} \dot{V}_2(t) &= s^T B^{-1} \dot{s} + \frac{1}{\gamma_{1I}} \tilde{K}_{1I} \dot{K}_{1I} + \frac{1}{\gamma_{1P}} \tilde{K}_{1P} \dot{K}_{1P} \\ &\quad + \frac{1}{\gamma_{1D}} \tilde{K}_{1D} \dot{K}_{1D} + \frac{1}{\gamma_{2I}} \tilde{K}_{2I} \dot{K}_{2I} + \frac{1}{\gamma_{2P}} \tilde{K}_{2P} \dot{K}_{2P} \\ &= s^T B^{-1} B(E\tilde{K} + u_S - \varepsilon) + \tilde{K}^T \Phi^{-1} \dot{\tilde{K}} \\ &= s^T (E\tilde{K} - \begin{bmatrix} \delta_1 \text{sgn}(s_1) \\ \delta_2 \text{sgn}(s_2) \end{bmatrix} - \varepsilon) - \tilde{K}^T \Phi^{-1} \Phi E^T s \\ &= -\delta_1 |s_1| - \delta_2 |s_2| - \varepsilon_1 s_1 - \varepsilon_2 s_2 \\ &\leq -(\delta_1 - |\varepsilon_1|) |s_1| - (\delta_2 - |\varepsilon_2|) |s_2| \\ &\leq 0. \end{aligned} \quad (23)$$

Using the adaptive PID controller (12) with the adaptive tuning laws (16), the inequality $\dot{V}_2(t) \leq 0$ can be obtained for nonzero value of the tracking error vector s . Since $\dot{V}_2(t)$ is a negative semidefinite function [i.e., $V_2(t) \leq V_2(0)$], it implies that s and \tilde{K} are bounded. Let the function $\Omega(t) \equiv$

$[(\delta_1 - |\gamma_1|)|s_1| + (\delta_2 - |\gamma_2|)|s_2|]$, and the following inequality is obtained from (23):

$$\int_0^t \Omega(\tau) d\tau \leq V_2(0) - V_2(t). \quad (24)$$

Because $V_2(0)$ is bounded and $V_2(t)$ is bounded and nonincreasing, thus the following inequality can be deduced:

$$\lim_{t \rightarrow \infty} \int_0^\infty \Omega(\tau) d\tau < \infty. \quad (25)$$

Meanwhile, as far as s is bounded, by using (13), it is obvious to realize that \dot{s} is also bounded. Then, $\Omega(t)$ is uniformly continuous. By using Barbalat's lemma [27], it can be shown that $\lim_{t \rightarrow \infty} \Omega(t) = 0$. Therefore, $s \rightarrow 0$ as $t \rightarrow \infty$. Consequently, the adaptive PID control system is asymptotically stable even if there exist the motor parameter variations and external load torque disturbances. ■

Remark 2: The angular acceleration (β) is normally not available. This angular acceleration can be estimated by the extended state observer [19], [28]. However, these estimations require accurate knowledge about some system parameters, so the algorithm seems to be complex and the accuracy of estimated values is highly sensitive to parameters variations. In this paper, β is simply computed by using the relation $\beta = \dot{\omega}$ as [29]. Moreover, this computation is independent of the system parameters:

$$\hat{\beta}(k) = \frac{\varphi}{T + \varphi} \hat{\beta}(k-1) + \frac{1}{T + \varphi} [\omega(k) - \omega(k-1)] \quad (26)$$

where φ is a sufficiently small filter time constant to limit the vulnerability of this computation to noise.

Remark 3: It should be noted that the proposed adaptive PID control strategy can be applicable to various electrical systems which have the mathematical form as (9). The overall design procedure of the proposed control scheme can be summarized as follows.

- Step 1:* Choosing the initial values of the PID gains by using the pole placement method [30] and [31].
- Step 2:* Constructing the decoupling control term u_f as (8) and the supervisory control term u_S as (19).
- Step 3:* Making the PID control term u_{PID} as (11) with the adaptation laws as (16).
- Step 4:* Giving the desired controller (12) by combining the three control terms in (8), (11), and (19).

Then, Fig. 1 shows the flow chart of the proposed adaptive PID control algorithm that represents the design procedure described in Remark 3.

IV. EXPERIMENTAL EVALUATION

A. Drive System Setup

In this section, the effectiveness of the proposed adaptive PID control scheme is evaluated through conducting a series of experiments on a prototype 1-hp SPMSM drive using TMS320F28335 digital signal processor (DSP), which is extensively used at ac motor drives. Note that with the advanced development of the DSP today, the proposed scheme with the simple adaptive tuning laws does not affect much the

TABLE II
TWO STUDY SCENARIOS FOR THE EXPERIMENT

Scenarios	Conditions	Details	Parameter Variations
1	Step change of the load torque	$\omega_d = 251.3$ rad/s; $T_L: 2.4 \text{ N}\cdot\text{m} \rightarrow 0 \text{ N}\cdot\text{m}$	R_s, L_s, J , and B are changed as +100, -30%, +120%, and +50%, respectively
2	Step change of the desired speed	$\omega_d: 251.3$ rad/s \rightarrow 125.7 rad/s; $T_L = 1 \text{ N}\cdot\text{m}$	

an online-tuning rule which can be adapted to the variations of any system parameters. Table II summarizes the two different scenarios described previously to assess the proposed algorithm.

Notice that it is not easy to directly change the motor parameters in experiments even if these values in simulations can be easily changed. As an alternative to implement the motor parameter changes in a real SPMSM drive, it can be done simply by changing the parameters in the control scheme. Therefore, in this paper, the changes of the system parameters in the controller have been made instead of changing the real system parameters in the SPMSM in order to experimentally verify the control performance of the proposed method and conventional PID method under the variations of some motor parameters (R_s , L_s , J , and B).

Since the conventional PID controller (10) possesses the control structure similar to the proposed adaptive PID controller (12), therefore, it is also implemented for the competitive comparison. Notice that the gains of the conventional PID controller are determined by the tuning rules of [30] and [31] based on the pole placement technique. In this paper, the conventional PID control gains are chosen as $K_{1P} = 30\,000$, $K_{1I} = 3000$, $K_{1D} = 100$, $K_{2P} = 200$, and $K_{2I} = 50$. Note that, these values are also used as the initial values to online tune the control gains of the proposed adaptive PID controller. In (16), the positive learning rates ($\gamma_{1P}\gamma_{1I}$, γ_{1D} , γ_{2P} , and γ_{2I}) should be sufficiently large to guarantee a fast learning process and small time to converge. However, if they are selected to be too large, the proposed adaptive PID algorithm may become unstable. Therefore, these values are chosen as $\gamma_{1P} = \gamma_{1I} = \gamma_{1D} = \gamma_{2P} = \gamma_{2I} = 0.1$ based on the fast learning process and system stability. Besides, the positive constants in the supervisory control term are chosen via extensive simulation studies as $\delta_1 = 5$ and $\delta_2 = 1$. Note that the angular acceleration (β) obtained from (26) is utilized in the conventional PID control scheme.

C. Experimental Results

Scenario 1: Under this scenario, the desired speed (ω_d) is set to 251.3 rad/s and the load torque (T_L) suddenly changes from 2.4 to 0 N·m under system parameter variations given in Table II. Figs. 3 and 4 present the comparative experimental waveforms of the proposed adaptive PID controller and the conventional PID controller, respectively. In detail, Figs. 3(a) and 4(a) show the desired speed (ω_d), rotor speed (ω), and speed error (ω_e), whereas Figs. 3(b) and 4(b) show the d -axis stator current (i_{ds}) and q -axis stator current (i_{qs}), respectively. It can be inferred from Figs. 3 and 4 that the regulation performance of the

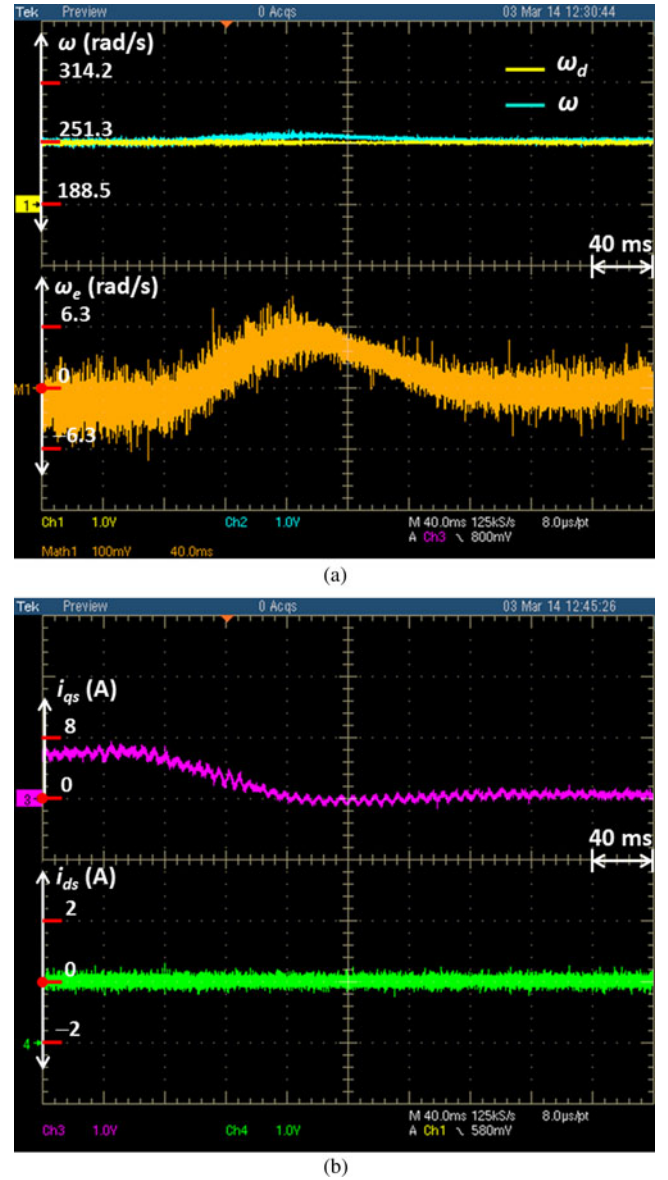
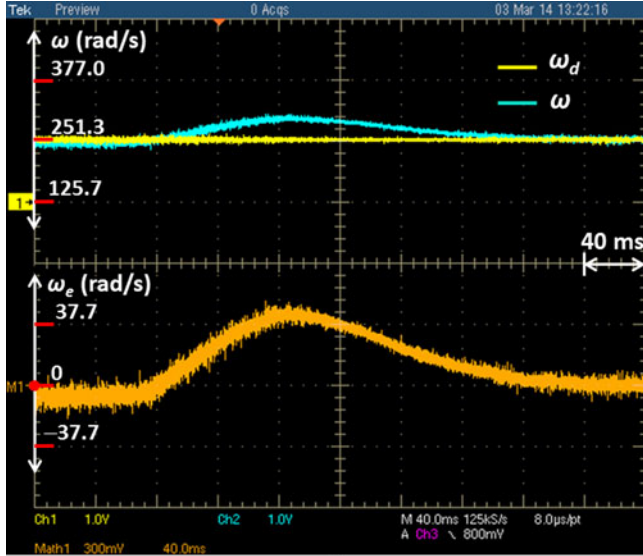


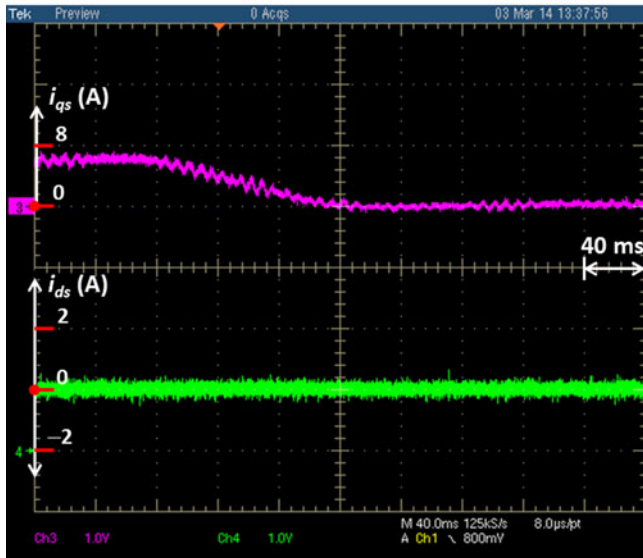
Fig. 3. Experimental results of the proposed adaptive PID control method when the load torque suddenly changes under system parameter variations. (a) Desired speed (ω_d), rotor speed (ω), and speed error (ω_e); (b) d -axis stator current (i_{ds}) and q -axis stator current (i_{qs}).

conventional PID control system is significantly improved after applying the adaptive tuning laws. That is, the proposed control scheme precisely tracks the desired speed with fast dynamic response (settling time: 196 ms) and small steady-state error (2.0%), under a sudden change in load torque. On the contrary, it is obvious from Fig. 4 that the conventional PID controller still shows its poor capacity when the load torque changes with a step, i.e., the settling time and steady-state error are 240 ms and 6%, respectively. It should be noted that the gains of the conventional PID controller are tuned under nominal parameters via extensive simulation studies. As shown in Fig. 4, its steady-state error is quite high because it lacks an adaptive capacity under parameter uncertainties.

Scenario 2: In this experimental scenario, the desired speed (ω_d) is suddenly changed from 125.7 to 251.3 rad/s and the load



(a)

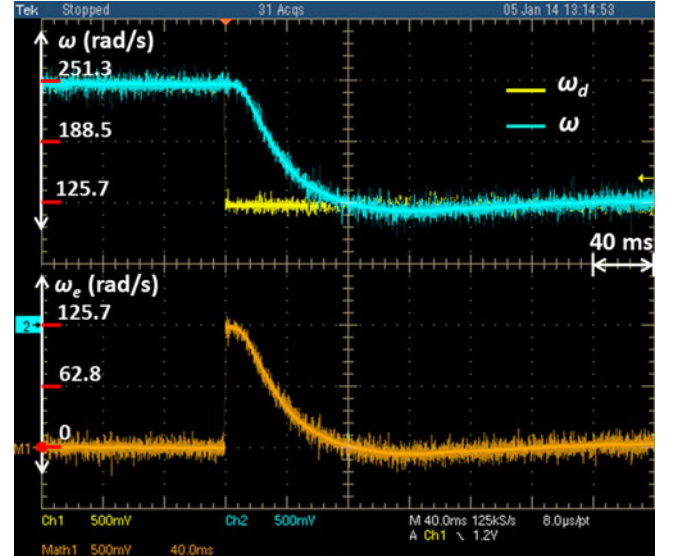


(b)

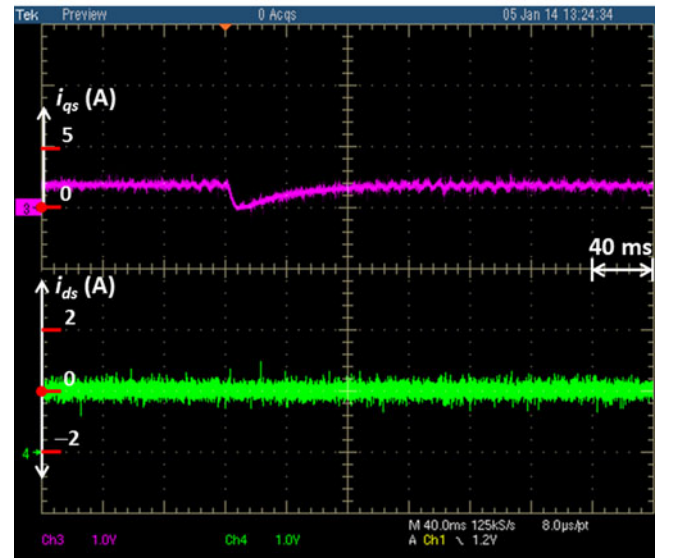
Fig. 4. Experimental results of the conventional PID control method when the load torque suddenly changes under system parameter variations. (a) Desired speed (ω_d), rotor speed (ω), and speed error (ω_e); (b) d -axis stator current (i_{ds}) and q -axis stator current (i_{qs}).

torque (T_L) is set at 1 N·m under system parameter variations. Figs. 5 and 6 show the experimental results of the proposed PID control method and the conventional PID control method. In Fig. 5, the rotor speed can be tracked to accurately follow the desired value (steady-state error: 1.6%). On the other hand, it can be seen in Fig. 6 that the conventional controller tracks the rotor speed with a considerable steady-state error (9.1%). In these figures, the proposed adaptive PID control scheme (settling time: 90 ms) exhibits the faster dynamic behavior than the conventional control method (settling time: 216 ms) under the parameter uncertainties.

The detailed comparative performance of the two control methods is summarized in Table III. From Figs. 3–6 and Table III, it is apparent that the proposed adaptive PID



(a)



(b)

Fig. 5. Experimental results of the proposed adaptive PID control method when the desired speed suddenly changes under system parameter variations. (a) Desired speed (ω_d), rotor speed (ω), and speed error (ω_e); (b) d -axis stator current (i_{ds}) and q -axis stator current (i_{qs}).

TABLE III
COMPARISON PERFORMANCES OF TWO CONTROL METHODS
IN THE EXPERIMENT

Scenarios	Control methods	Proposed adaptive PID controller		Conventional PID controller	
		Settling time (ms)	Steady-state error (%)	Settling time (ms)	Steady-state error (%)
1		196	2.0	240	6.0
2		90	1.6	216	9.1

controller can more effectively improve the control performance (i.e., faster dynamic response and smaller steady-state error) than the conventional PID controller when there exist the motor parameter variations and external load disturbances.

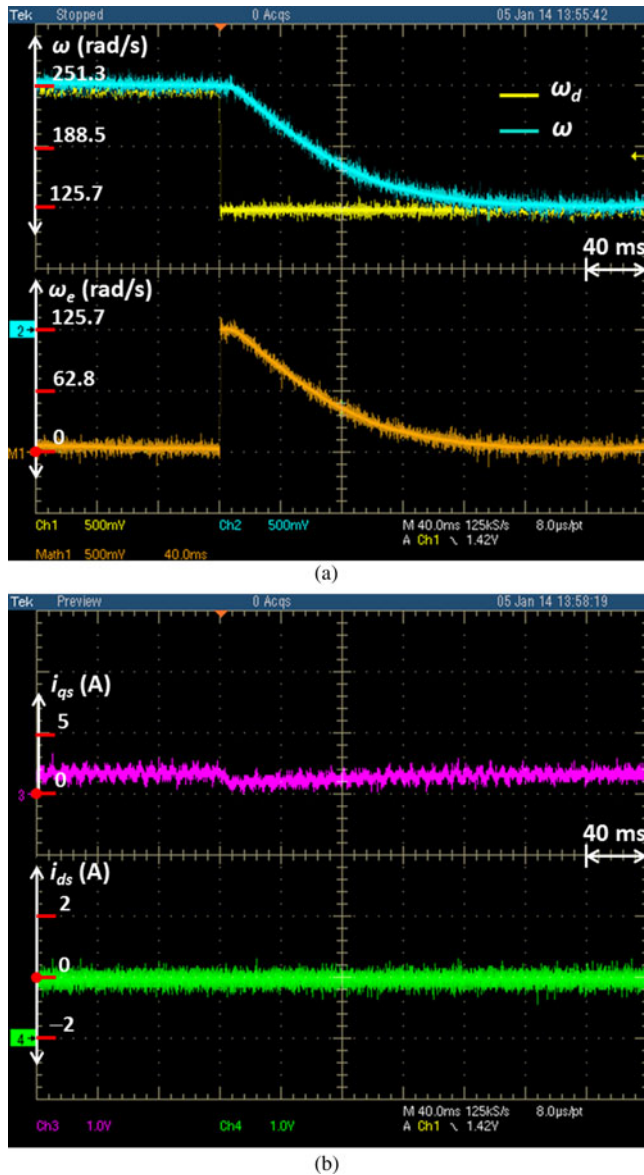


Fig. 6. Experimental results of the conventional PID control method when the desired speed suddenly changes under system parameter variations. (a) Desired speed (ω_d), rotor speed (ω), and speed error (ω_e); (b) d -axis stator current (i_{ds}) and q -axis stator current (i_{qs}).

V. CONCLUSION

In this paper, an adaptive PID control method for speed control of the PMSM drives was proposed. The proposed control algorithm was simple and easy to implement in the practical applications. By using the gradient descent method, the adaptive tuning laws were proposed to autoadjust the PID gains that can achieve favorable tracking performance. Therefore, the proposed control scheme could guarantee the accurate and fast speed tracking in spite of the system parameter variations and external load disturbances. Moreover, the stability analysis of the proposed control system was described in detail. To verify the effectiveness of the proposed control strategy, the experiments were conducted. For comparison purpose, the conventional PID controller was also tested at the same conditions as the proposed controller. It was evidenced through the

experimental results that the proposed adaptive PID control algorithm could considerably enhance the control performance compared to the conventional PID control method. As a result, the main accomplishments of this paper are summarized as follows:

- 1) it provided a novel adaptive PID control strategy with a detailed design procedure;
- 2) it offered the mathematical proof about the stability and zero convergence of the control system with Lyapunov's direct method and related lemmas;
- 3) it tested the adaptive PID control scheme that can precisely track the speed of the SPMSM drive under motor parameter variations and external load disturbances;
- 4) it presented the experimental results of the conventional PID controller for comparative studies.

Nowadays, the research activities are going on to develop the new analysis and tuning methods for the PID gains, so the proposed adaptive PID control scheme can help reduce the difficulties in these issues.

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